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# **Relevance of Comparative Approaches Plan Critical and Integral**

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**ABSTRACT**: The work in this paper is to compare relatively recent fatigue criteria with already proven and recognized formalisms used for fatigue design calculations in industry. We dispose, to do this, of a large number of multiaxial tests corresponding to solicitations very diverse put to contribution to validate the criteria. Three criteria of integral approach (Zenner, Fogué and Papuga I), three criteria of approach critical plan (Dang Van, Robert and Papuga II) and two criteria based on the invariants of the constraints (Crossland and Vu) are retained. Two analyzes were carried out:

- The one on the accuracy of the predictions of the criteria, from a general point of view on all the tests used in this work. It emerges from this analysis that the integral approach criteria Zenner and Fogue are practically the most accurate;
- The second analysis consists in showing the comparative relevance of the critical plan and integral approach to translate the behavior into multiaxial fatigue. A representative of each approach is chosen, the criterion of Dang Van, for the critical plan approach, and that of Fogué, for the integral approach. The analysis is based on the distribution of their damage indicator by plane, Eh, and on the value of their fatigue function, E, for the test considered. The selected tests are representative of the main cases of solicitations that can be encountered. This study allowed to show the influence of the middle part and the phase shift of the solicitations as to the forecasts of the two criteria (fatigue functions). This is the integral approach that takes precedence over other approaches developed.

**KEYWORDS:** Criteria, Multiaxial fatigue, Damage indicator, Fatigue function, Average stress influence, Phase shift influence.

# I. INTRODUCTION

One of the specific problems of the fatigue design of the mechanical components is taken into account of the character multiaxial. The tool adapted to deal with this aspect is a criterion of multiaxial fatigue, which establishes a relationship between the stresses of multiaxial cycle analysed and fatigue characteristics of the material, relationship that allows to adjudicate on the fatigue strength of the material when subjected to stress cycle. Practically, the fatigue analysis is carried out in all the points where of the component are known the stresses. If the critical point of the component is located in advance, fatigue calculations are performed only in this point. In the opposite case, the calculations are carried out fatigue at all the points to determine the critical area of the component from the point of view of fatigue and thus establish a mapping life of this component. The work presented here have had the ambition provide the Design Office elements decision as to regarding criteria multiaxial fatigue. New proposed criteria have appeared in recent years [1-11]. Some cycles of specific constraints prove discriminant overlooked the most efficient criteria and thus allowing to better perceive the specificity of the criteria [12] and [11].



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#### II. GENERAL FORMALISM CRITERIA MULTIAXIAL FATIGUE

Order to rule on a given compared to the endurance limit of the material cycle multiaxial, the principle implemented by a fatigue criterion is the constitution of a fatigue function denoted E, which connects the parameters of stresses specific to multiaxial cycle analyzed and endurance limits of the material as  $\sigma_{-1}$ ,  $\tau_{-1}$ ,  $\sigma_{0}$ , etc. The function of fatigue is equal to unity when the endurance limit of the material is attained through analyzed the cycle; this relationship is written as:

$$E(t) = E([\sigma(t)], \sigma_{-1}, \sigma_0, \tau_{-1})$$
(1)

Where  $[\sigma(t)]$  is the stress tensor of the cycleduring aperiod, Table 1 gives the different expressions of the indicator of damage and functions studied fatigue of the criteria [5, 7, 11-13].

Author	Formalism	Wedging					
Based oninvariantsstresscriteria							
Crossland	$E_{CR} = \frac{\sqrt{J_{2a}} + A.I_{1max}}{B}$	σ-1, τ-1					
Vu	$E_{VU} = \frac{1}{\beta} \underset{t}{Max} \left( \sqrt{\gamma_1 J_{2a}^2(t) + \gamma_2 J_{2m}^2 + \gamma_3 I_f(I_{1a}, I_{1m})} \right)$	σ-1, τ-1					
	Criteria for critical plan approach						
Dang Van	$E_{h}(t) = \frac{\left\  \overline{\tau_{ha}(t)} \right\  + \alpha p(t)}{\theta},  E_{DV} = \underset{t,h}{Max} \left[ E_{h}(t) \right]$	$\sigma_{\text{-1}},\tau_{\text{-1}}$					
Robert	$E_{h}(t) = M_{t} \left\{ \frac{\tau_{ha}(t) + \alpha \cdot \sigma_{hha}(t) + \beta \cdot \sigma_{hhm}}{\theta} \right\}, E_{RB} = M_{h} \left[ E_{h}(t) \right],$	$\sigma_{\!-\!1},\tau_{\!-\!1,}\sigma_{\!0}$					
Papuga II	$E_{h} = \frac{1}{\sigma_{-1}} \sqrt{\alpha \tau_{ha}^{2} + \beta \left(\sigma_{hha} + \frac{\tau_{-1}}{\sigma_{0}} \sigma_{hhm}\right)} E_{PAII} = Max_{h}(E_{h})$	$\sigma_{\!-\!1},\tau_{\!-\!1}$					
Criteria Integralapproach							
Fogue	$E_{h} = \frac{a\tau_{ha} + b\sigma_{hha} + d\sigma_{hhm}}{\sigma_{-1}}, E_{FG} = \sqrt{\frac{1}{S}\int_{S} E_{h}^{2} ds}$	$\sigma_{\!-\!1},\tau_{\!-\!1,}\sigma_{\!0}$					
Papuga I	$E_{h} = \alpha \tau_{ha}^{2} + \beta \left( \sigma_{hha} + \frac{\sigma_{-1}}{\sigma_{0}} \sigma_{hhm} \right), E_{PAI} = \sqrt{\frac{1}{S} \int_{S} E_{h} ds}$	$\sigma_{-1}, \tau_{-1}$					
Zenner and Liu	$E_{h} = a\tau_{ha}^{2}(1 + m\tau_{hm}^{2}) + b\sigma_{ha}^{2}(1 + n\sigma_{hhm}), E_{Z} = \frac{1}{\sigma_{-1}}\sqrt{\frac{1}{S}} \frac{1}{S} E_{h} ds$	$\sigma_{\!-\!1},\tau_{\!-\!1}$					

### TABL 1 -FORMALISMANDWEDGING OFCRITERIA

#### **III.ANALYSIS PREDICTIONS MULTIAXIAL FATIGUE CRITERIA**

We go here compare the proposals of relatively recent formalisms already proven and accepted criteria for fatigue, and used for the calculations of the fatigue design in industry (Dang Van criteria and Zenner). We have to do this, a large number of tests multiaxial corresponding to very different solicitations enlisted to validate the criteria. This validation is therefore of a very general character since it the criteria allows confront the criteria of fatigue to a large panel of different conditions of use in terms of solicitations with or without phase shifts, with or without the presence



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of mean stresses at levels loading corresponding to lifetimes variant of 105 to 2x106 cycles and quite different from each other steel grades. An initial survey of tests are listed into 17 series had been carried out by B. Weber, a 18th series of tests came to complete this database, a total of 259 trials multiaxiaux [NGA10].

### III.1. PRINCIPEOF VALIDATIONTHE CRITERIAMULTIAXIALS

Each series of tests corresponds to the multiaxial fatigue limit of the material for a given number of cycles. In this life are also known fatigue limits of the material under simple sollicitations( $\sigma_{-1}$ ,  $\tau_{-1}$ ,  $\sigma_0$ ). For these simple calibration tests, the criterion is verified by definition (E=1). To adjudicate the validity of a criterion, one examines the gap recorded between the actual function fatigue E of the criterion and its theoretical value equal to unity. The validity of the criterion for a given test is judged by the amount denoted  $\Delta E$  and equal to E-1. The value of  $\Delta E$  allows to the passage to situate the multiaxial cycle tested compared with the fatigue limit of the material:

- 1. if  $\Delta E < 0$ , the cyclemultiaxial within the meaning of the criterionisless than the fatigue limit of the material, which means that the testprovides for a longer lifegreater than that obtained experimentally. Such cases penalizes the StudyOffice, as it means a non-conservative prediction of service life, thus keeping the component;
- 2. if  $\Delta E = 0$ , the test correctly predicts that the fatigue limit is reached. It turns out in this case therefore exact ;
- if∆E>0, the fatigue criterion perceives the multiaxial cycle analyzed as superior capabilities fatigue resistance of the material, it provides for therefore a shorter service lifethan is actually obtained for this cycle there that is to say an earlier crack initiation than it is reality. It provesso in this case conservative for the design office.

### **III.2. RESULTS**

To give the synthesis of all theresults obtained by a fatigue criterion on the database of the tests in its totality, we construct a histogram of the results reported in which the ordinate, the number of tests with a standard  $\Delta$ E belonging to a given interval (class of belonging). If the criterion was perfect all tests should be in the middle of the histogram (for a difference of validity  $\Delta$ E zero). The trends to a conservative or nonconservative forecast translate a displacement of the histogram areas where  $\Delta$ E >0 and  $\Delta$ E <0, respectively. Figure 1 shows the histogram sobtained for the 3 criteria of integral (Zenner, Fogue and PapugaI), the 3 criteria critical planeapproach (Van Dang, Robert and PapugaII) and 2 criteria based on invariant stresses (Crossland and Vu).



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Fig.1 -Histogramsofvalidity of the criteriastudied

# **III.2.1. ANALYSIS OF RESULTS**

The analysis will focus on the prediction accuracy of the criteria, this from a general point of view on all tests contribution set to in this work. For this purpose, Tab.2 and Fig.2 provides a summary of the results achieved with the 8 fatigue criteria studied, confronted with the databank of 259 tests multiaxials fatigue.



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		IAD.2 -0	UMMAALOFA	LOULIS			
	aracter	С	haracter	Appro	opriateness		
	no co	nservative	con	nservative	of the	e criterion	
	ΔΕ	. < 0 %	Δ	E > 0 %	$-5\% \leq \Delta E \leq +5\%$		
Authors	Number	Percentage	Number	Percentage	Number	Percentage	
Zenner (ZN)	131	50,58 %	128	49,42 %	118	45,56%	
Papuga-I (PAI)	68	37,84 %	161	62,16 %	108	41,70%	
Papuga-II(PAII)	170	65,64 %	89	34,36 %	122	47,10%	
VU (VU)	156	60,23 %	103	39,77 %	111	42,86%	
Fogue (FG)	121	46,72 %	138	53,28 %	117	45,17%	
Crossland (CR)	35	13,51 %	224	86,49 %	58	22,39%	
Dang Van (DV)	138	53,28 %	121	46,72 %	91	35,14%	
Robert (RB)	139	53,67 %	120	46,33 %	101	39,00%	
100	Justesse (-5% ≤ ∆E ≤ +5%)	100	Non conservatif ( $\Delta E < 0\%$ )	Conse	ervatif (∆E > 0%)		
90 80 70 60 40 20 10 0	ZN EPAI EPAI VU FG CR	90 80 70 60 50 40 30 20 10 0 V R8 ZN	EPAI EPAI VU FG CR	90 80 70 60 90 80 90 80 90 80 90 80 90 80 90 80 90 80 90 80 90 80 90 80 90 80 90 80 90 80 90 80 90 80 90 80 90 90 90 90 90 90 90 90 90 90 90 90 90	NI VU FG CR DV	RB	

# **TAB.2 - SUMMARYOFRESULTS**

Fig.2 -Histogramsthe synthesis ofcriteria studied

As fromsummary table (Tab.2) and histogramsvaliditycriteria(Fig.2) above, the analysis and evaluation of results obtained areas follows :

- a. The integral approach Zenner and Fogue criteria are practically the most accurate. They feature formalisms that predispose to better detect whether endurance of metallic materials threshold is reached or not. The forecasts are quite balanced and practically no uncommanded or opposite excess conservatism.
- b. The thirdcriterionintegral approach, that of PapugaI, is a little lessefficientbut proves to bemostlyquite stronglyconservative.
- c. The criterion of critical planeapproach Papuga II gives excellent predictions (slightly better than integral approaches Zenner and Fogue) but it turns out heis very strongly non-conservative.
- d. The other twoapproachescriticalplane typestudied (Dang VanandRobert) are less efficient thanthe integral approachandconservativetrendrathernot.
- e. Giventhecriterion(based oninvariants) is almoston a par with the integral approachin terms of the prediction, but it turns out to be non-conservative.
- f. The criterion of Crossland (invariant constraints) turns out to be rather mediocrecompared to other criteria. It reveals more conservative.

# IV.PARTICULARITY APPROACHES CRITICAL PLANE AND INTEGRAL

With the aim to analyze the relevance of comparative approaches critical and integral plane translate behaviormultiaxial fatigue, we chose a representative of each approach, the Dang Van criterion for the approach to critical level and that of Fogué for the integral approach. The analysis is made on the distribution of the indicator of damage by plane Eh and the function of the fatigue test considered. Successful tests are representative of major cases of solicitations may be encountered. Five stress cycles in total are used for this analysis [11]. The five stress cycles used correspond either to the state of constraint proportional either to the non-proportional stress state. In Figures 3 to 7, are shown on the left, the components of the selected stress cycles. To the right of these figures, for the purpose of comparing the two multiaxial fatigue approaches, are shown the distributions of the damage indicator by  $E_h$  plane.  $E_{DV}$  and  $E_{FG}$  are respectively the damage indicators Dang Van criteria and Fogué for the successful stress cycle.



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#### IV.1. CYCLES CORRESPONDING TO STATES PROPORTIONAL STRESS

Twocyclestoproportionalstressstateswere selected forthis analysis.

• Test 1 (I04):tractionbalancebiaxialphase, fixedprincipal directions



Fig.3- CycleTest.1 and distributions of damage indicators

The two criteriaare moreaccurate from the perspective of the value of their respective function of fatigue. The Dang Van criterion how everbetter than that of Fogue for this test, its function fatigue being closer to the unit value. The distribution of the indicator Ehthis for two criteriaa multitude of critical planes defined by  $\gamma = 45^{\circ}$  or  $\gamma = 135^{\circ}$  to the Dang Van criterion and  $\gamma = 55^{\circ}$  or  $\gamma = 125^{\circ}$  for those of Fogue.

• Test 2(B11): Traction-torsionphase, fixedprincipal directions

•



Fig.4- CycleTest 2and distributions of damage indicators

The two criteriashall rulein a wayalmost similar to the severity of the stress state in terms of the fatigue resistance, but the criterionFogueishowevermore accurate(closer to the unitvalue) than that of Dang Van. The distribution of this indicator plancritical coordinates ( $\gamma = 90^{\circ}$ ,  $\varphi = 55^{\circ}$ ) to the criterion Fogue, and two critical coordinate planes ( $\gamma = 90^{\circ}$ ;  $\varphi = 70^{\circ}$  or  $160^{\circ}$ ) for the Dang Van criterion.



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#### IV.2. CYCLES CORRESPONDING TO STATES OF NO-PROPORTIONAL STRESS

The same type of analysis is conducted here for three cycles states stress non-proportional, fixed or mobile principal directions.

Test 3(I07): Traction biaxialphase, fixed principal directions



Fig.5- CycleTest 3and distributions of damage indicators

The two criteriaare veryfatiguedistant from real behavior of the material for this test(non-conservative predictions). The distribution of the indicator criterion Fogue presents two critical coordinate planes ( $\gamma$ =55° or 125°;  $\phi$ =90°), while that obtained for by Dang Vanpresenta multitude of coordinates  $\gamma$ =45° and  $\gamma$ =135°.

• Test 4 (B04) : Traction-compression out of phase, fixedprincipal directions



Fig.6 -CycleTest 4and distributions of damage indicators

In view offunction valuesoffatigue, the criterion of Fogueis much betterin this test than that of Dang Van. The distribution of the indicator presents two critical planes for each of two criteria, the coordinate ( $\gamma = 90^\circ$ ; = $\phi 60^\circ \circ r 120^\circ$ ) and those of Fogue ( $\gamma = 90^\circ$ ; = $\phi = 45^\circ \circ r 135^\circ$ ) to those of Dang Van.



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Fig.7- CycleTest5and distributions of damage indicators

Bothcriteria areratherefficientfor this test, the criterion of Foguebeing somewhat better than that of Dang Van. The distribution of the indicator has a single critical plane for both criteria whose coordinates are ( $\gamma = 90^{\circ}$ ;  $\phi = 175^{\circ}$ ) according to the criterion Fogue and ( $\gamma = 90^{\circ}$ ;  $\phi = 80^{\circ}$ ) according to the Dang Van.

Fig.8 shows the EFG and EDV fatigue function values of the criteria for the five tests.



Fig.8 – Recapitulation of the functions fatigue criteria Fogue and Dang Van for five tests

### V. INFLUENCE OF MEAN STRESS AND PHASE DISPLACEMENT ON THE FORECASTS OF THE TWO CRITERIA

An additional study on the influence of the mean stress and phase displacement solicitations regarding forecasts the two criteria (functions fatigue) has identified the following observations:

### V.1. SENSITIVITY TO MEAN STRESS

The averagetensile stresshas a detrimental effecton fatigue strength of thematerial according to two criteria. Sensitivity to mean stressis more pronounced for the criterion Fogue compared to the Dang Van criterion. This sensitivity is translated by the value of the function of fatigue (for the test 1) passing of 0.60 to 1.063 for the criterion Fogue and 0.76 to 1.015 for the Van Dang, respectively without and with biaxial averages constraints of the test. The same trend is observed for the test 3 but is less pronounced because an average tensile strain is superimposed on an average compressive stress in a perpendicular direction.

### V.2. SENSITIVITY THE PHASE SHIFF

The twocriteriaarealsosensitivethe	phase	shift.	For	test4,	thevalue	of	the	functionfatiguewithout	phasepasses
from 0.867 to 0.966 with phase	shiftfor			the			criterionFogueand0.902without		



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phase shiftto0.793 with phase shift or the Dang Van criterion. The phase shift therefore renders more solicitations endommage antes within the meaning offatigue resistance for the criterion. For tension-torsion test 50 ut of phase, the deletion of phase shift between the components for the solicitation function increases the fatigue of the two criteria. That of Fogue passes from from 1.042 to 1.055 and the Dang Van varies from 1.056 to 1.142. For this type of solicitation, therefore the phase shift as beneficial effect fatigue where a straction compression this is true only for the Dang Van criterion.

#### VI. DISCUSSION

The two points of discussion of the results obtained carry firstly on the character overall conservative or nonconservative criteria on all of tests of database, independently of any consideration of the nature of the tests, and secondly the accuracy of the predictions criteriathere too a general point of view on all put to test in this work. For this purpose, Tab.2 and Fig.2 providing a review of results obtained with 8 fatigue criteria studied, confronted with the database of 259 multiaxial fatigue testing.

From previous table summary and histograms validity criteria that were presented, the analysis and stock of the results obtained are as follows:

- a. The integral approach Zenner and Fogue criteria are practically the most accurate. Formalisms predispose their best to detect if the of metallic materials endurance threshold is reached or not. Forecasts are quite balanced and practically no very marked conservatism or reverse excess.
- b. 2. The third criterion integral approach, that of Papuga I, is a little less efficient but proves to be mostly quite strongly conservative.
- c. The criterion of critical plane approach of Papuga II gives excellent predictions (slightly better than integral approaches Zenner and Fogue) but it turns out it is strongly non-conservative.
- d. The other two approaches critical plane type studied (Dang Van and Robert) are less efficient than the integral approach and conservative trend rather not.
- e. The criterion VU (based on invariants) is almost at par with the integral approach in terms of prediction, but it turns out to be non-conservative.
- f. The criterion of Crossland (invariant constraints) turns out to be rather mediocre compared with the other criteria. It reveals more conservative.

#### VII. CONCLUSION

The integral approach seems to take precedence over other approaches developed for the criteria of multiaxial fatigue, and seems to justify industrial practice most frequently observed in Germany. The benefit a little back of the Dang Van criterion, though very commonly used in France, may be to find in the fact that his "critical plane" character is not respected quite at the foot of the letter : the combination of the hydrostatic pressure which is the normal stress on the existing plane octahedral, with the alternating shear stress acting on a given plane, in fact associated constraint parameters that are not related to the same physical plane (unless, and only in this case, when the plan is analyzed the octahedral plane, and that the latter "turns" not during the cycle). Assuming that the severity of loading on the material plane or facet is related to the constraints that actually generates the load on this plan it seems more logical to use the normal and tangential stresses specific to the relevant plane. Deficiency of the Dang Van criterion vis-a-vis the effect of the mean stress is related to the fact that the mean and alternating parts of the hydrostatic stress involved in the same proportion (coefficient  $\alpha$ ) since calibrated only on two fatigue limits in alternating tension and alternating torsion symmetrical, which do not translated the actual particular influence of mean stress.

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